



## Error-based mesh adaptation during crack propagation simulation with X-FEM

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### Context:

- ▶ Linear Elastic Fracture Mechanics
- ▶ Use of X-FEM
- ▶ Propagation of pre-existing crack under fatigue based on SIFs

Even in X-FEM, mesh refinement is needed for:

- ▶ Implicit representation of cracks
- ▶ Accurate computation of stress fields

### Solutions:

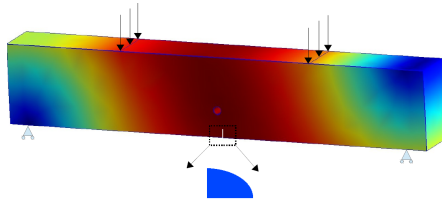
- ▶ Use fixed mesh with refinement in areas where cracks are expected
- ▶ Mesh adaptation
  - ▶ Engineering approach based on geometric parameters
  - ▶ Scientific approach based on error estimation

### During crack propagation:

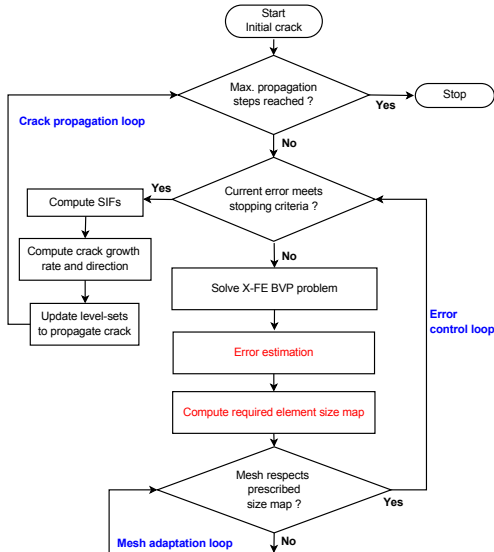
- ▶ Keep fine mesh along crack front
- ▶ Number of DOFs +/- constant

# Mesh adaptation during crack propagation

## An engineering approach



# Error control MA - Global strategy



1. Solution of the X-FEM problem:

$$\hat{\mathbf{u}}(\mathbf{x}) = \sum_{I=1}^n \phi_I(\mathbf{x}) \mathbf{a}_I + \sum_{J \in \mathcal{J}_h} \phi_J(\mathbf{x}) h(\mathbf{x}) \mathbf{b}_J + \sum_{K \in \mathcal{J}_T} \phi_K(\mathbf{x}) \left( \sum_{\ell=1}^4 F_{\ell}(\mathbf{x}) \mathbf{c}_{K\ell} \right)$$

2. eXtended Global Recovery (XGR) - build smoother approximation:

$$\hat{\varepsilon}(\mathbf{x}) = \sum_{I=1}^n \phi_I(\mathbf{x}) d_I + \sum_{J \in \mathcal{J}_h} \phi_J(\mathbf{x}) h(\mathbf{x}) e_J + \sum_{K \in \mathcal{J}_K} \phi_K(\mathbf{x}) \left( \sum_{\ell=1}^4 G_{\ell}(\mathbf{x}) f_{K\ell} \right)$$

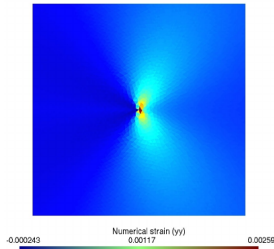
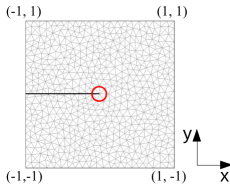
► Coefficients determined by minimization of  $\int_{\Omega} \| \hat{\varepsilon} - \varepsilon(\hat{\mathbf{u}}) \|^2 d\Omega$

3. Estimation of error in energy norm:

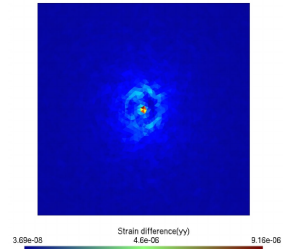
$$\| \hat{\varepsilon} - \varepsilon(\hat{\mathbf{u}}) \|_{\Omega} = \sqrt{\int_{\Omega} (\mathbf{C} : (\hat{\varepsilon} - \varepsilon(\hat{\mathbf{u}}))) : (\hat{\varepsilon} - \varepsilon(\hat{\mathbf{u}})) d\Omega}$$

# XGR - Application to the Westergaard problem

- ▶ Single edge crack subjected to pure mode I
- ▶ Analytical solution available
- ▶ Exact tractions applied on the boundary



$\hat{\epsilon}_{eq}$   
Recovered strain field



$\hat{\epsilon}_{eq} - \epsilon_{eq}(\hat{u})$   
Estimated local error  
(on strain)

Element refinement/coarsening:  $h_i^{\text{new}} = \xi_i h_i^{\text{current}}$

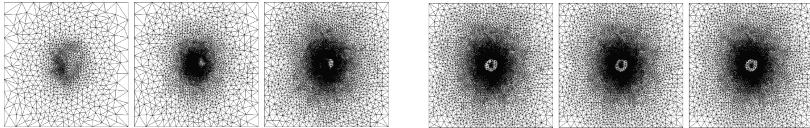
Definitions of  $\xi_i$ :

- ▶ Equal-distribution criterion: reduce global error + similar local error over all elements
  - ▶ Global error:  $\xi_i^g = \frac{\max(0.6\|\hat{\epsilon} - \epsilon(\hat{\mathbf{u}})\|_\Omega, 0.02\|\epsilon(\hat{\mathbf{u}})\|_\Omega)}{\|\hat{\epsilon} - \epsilon(\hat{\mathbf{u}})\|_\Omega}$
  - ▶ Local error:
    - ▶  $\xi_i^{\ell e} = \frac{\|\hat{\epsilon} - \epsilon(\hat{\mathbf{u}})\|_\Omega}{\sqrt{N}} \frac{1}{\|\hat{\epsilon} - \epsilon(\hat{\mathbf{u}})\|_i}$
    - ▶ Volume-weighted:  $\xi_i^{\ell v} = \|\hat{\epsilon} - \epsilon(\hat{\mathbf{u}})\|_\Omega \sqrt{\frac{\Omega_i}{\Omega}} \frac{1}{\|\hat{\epsilon} - \epsilon(\hat{\mathbf{u}})\|_i}$
  - ▶  $\xi_i = (\xi_i^g)^{\frac{1}{n}} (\xi_i^{\ell e})^{\frac{2}{2n+d}}$  - Global + local [Oñate *et al.*]
  - ▶  $\xi_i = (\xi_i^g \xi_i^{\ell e})^{\frac{1}{n}}$  - Global + local (simplified) [Zienkiewicz *et al.*]
  - ▶  $\xi_i = (\xi_i^g \xi_i^{\ell v})^{\frac{1}{n}}$  - Global + local (vol. weighted) [Oñate *et al.*]
- ▶ Min-Number criterion: Minimize number of elements for given rel. global error:  $\xi_i = \frac{\theta_0^{1/q}}{\theta_i^{1/(q+1)} \left[ \sum_i \theta_i^{2/(q+1)} \right]^{1/2q}}$  [Coorevits *et al.*]

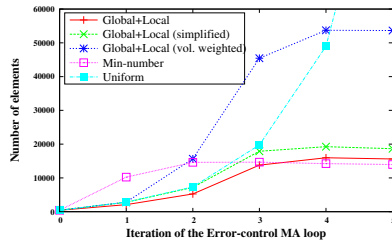
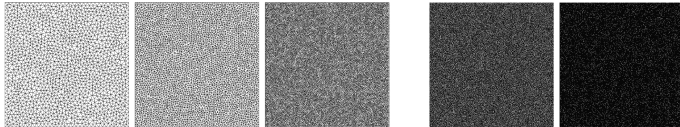
# Mesh evolution during error control loop

## Application to the 2D Westergaard problem

Global  
+ local  
(vol.  
weighted)



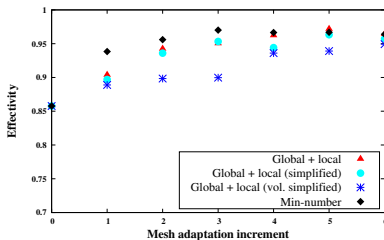
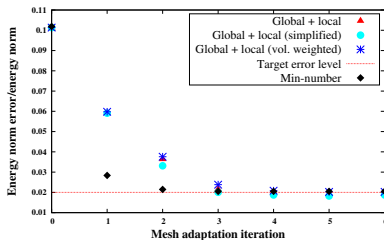
Uniform





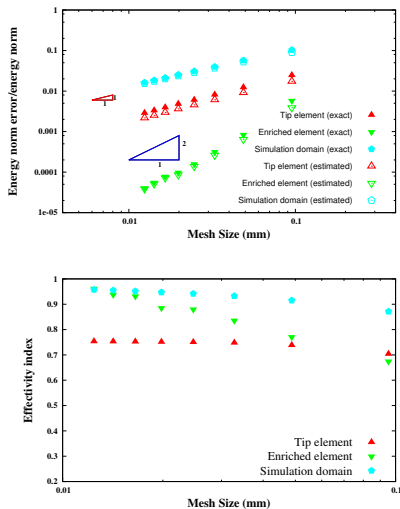
# Comparison of the different MA strategies

## Evolution of global Errors and effectivities

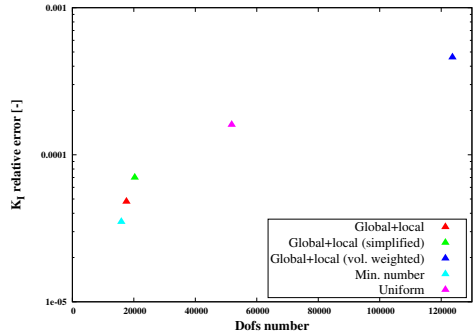
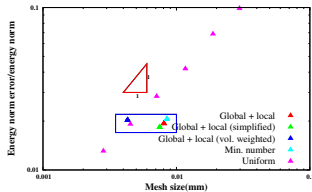


# Comparison of the different MA strategies

## Evolution of local Errors and effectivities



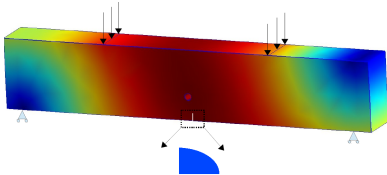
# Influence of mesh adaptation strategy on SIFs



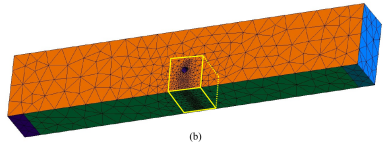
Similar energy error can give significantly different errors on SIFs

# Application to 3D crack propagation

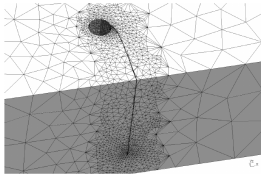
4-points bending test



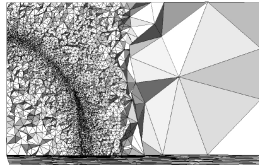
Initial mesh



Surface mesh (step 32)

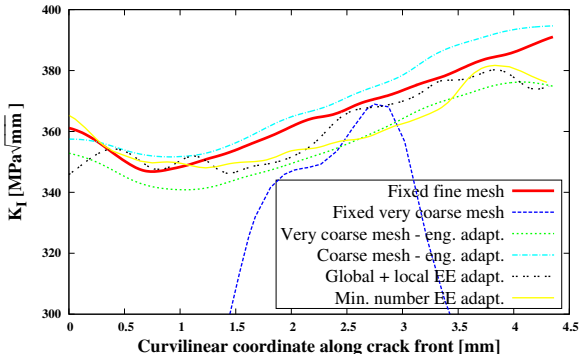


Mesh cross-section around crack front (step 32)

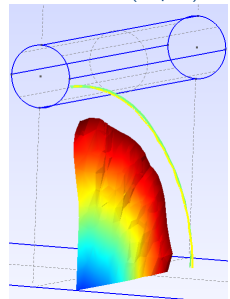


# Error controlled MA for 3D crack propagation

Sifs along crack cront (step 0)



Crack fronts (Step 30)



Crack growth rate:  $\frac{da}{dn} = C \Delta K_{eq}^{4.75} \rightarrow \text{Error on } K_i \text{ strongly ampl}$

# Rough comparison of the computational cost

	Fixed fine	Very coarse Eng.	Global + local	Min-number
Initial mesh [dofs]	883.000	11.400	3.300	3.300
Nbr. propa. steps	35	35	35	35
Nbr it Error est.	0	0	3	3
XGR [dofs]	N/A	N/A	1.200.000	420.000
X-FEM [dofs]	883.000	18.000	400.000	140.000
Time per step [s]	1000-2400	70-80	8.000-20.000	1.000-5.200

## Conclusions:

- ▶ Global + local & Min-number strategies give more reliable predictions
- ▶ Sub-optimal convergence rate in crack tip element
- ▶ Similar energy error can give significantly different errors on SIFs
- ▶ Our engineering approach is (surprisingly?) an extremely efficient solution

## Perspectives:

- ▶ Error-based mesh adaptation on initial configuration only
- ▶ Use recovery technique limited to a volume around the crack front to reduce XGR computation time
- ▶ Goal-oriented mesh adaptation is a must